

Select the best material for each of the requirements.

Property

Material

	A	B	C
Elastic modulus (ksi)	10,000	30,000	40,000
Yield strength (ksi)	30	300	-
Ultimate strength (ksi)	50	320	200
True fracture strength (ksi)	100	350	200
Strain hardening exponent	0.26	0.05	-
Reduction in area (%)	65	15	0.01
Density (lb/in ³)	0.28	0.25	0.40

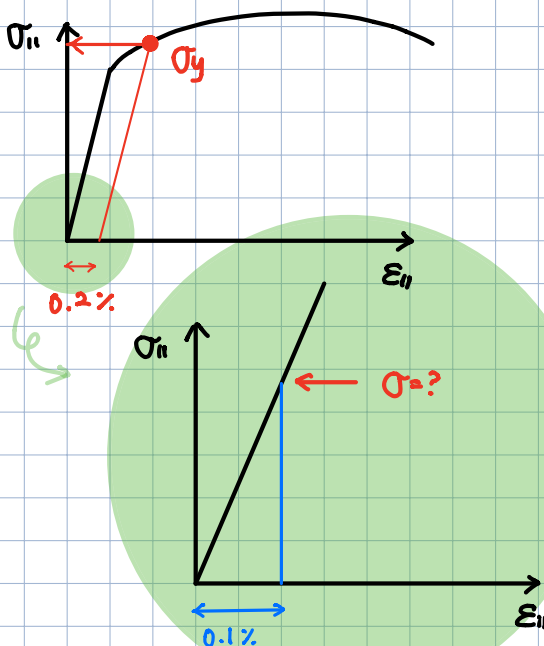
(a) Max. tensile load → σ_{UTS}

	σ_{UTS}	
A	50	
B	320	✓
C	200	

(b) Max. tensile load per pound on similar rods → $\sigma_{UTS}/\text{density}$

	σ_{UTS}	density	$\sigma_{UTS}/\text{density}$	
A	50	0.28	178.6	
B	320	0.25	640	✓
C	200	0.40	500	

(c) Max. tensile load at $\epsilon = 0.001$

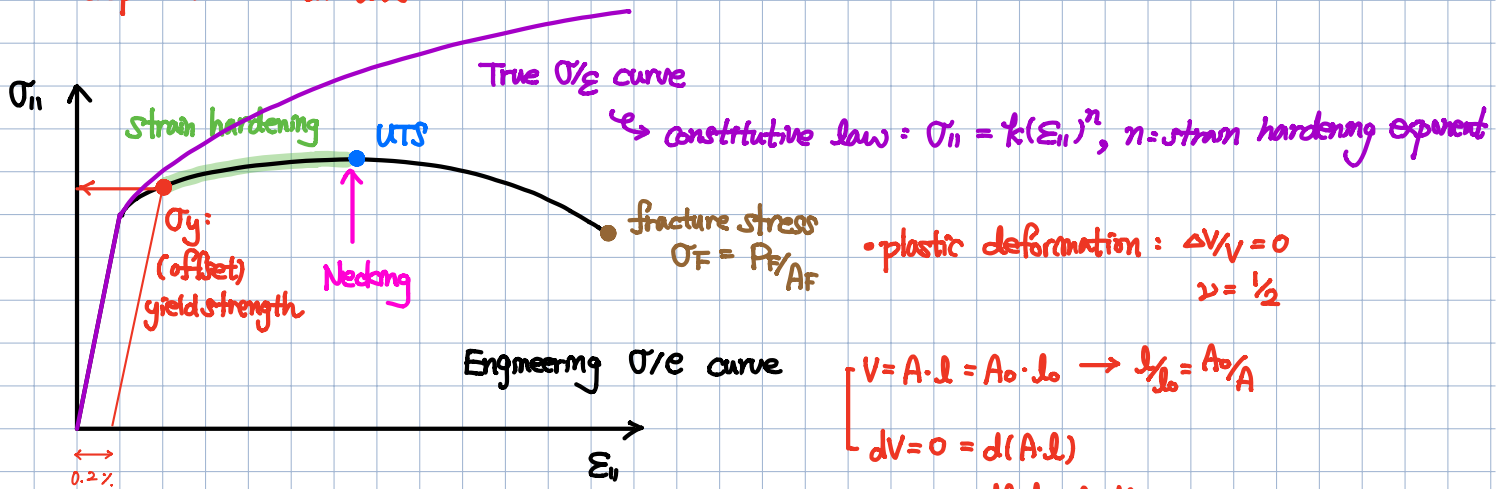


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	Elastic modulus	
A	10,000	
B	30,000	
C	40,000	✓

(d) Max. uniform elongation before necking

= displacement until UTS



Engineering σ^e/ϵ \longleftrightarrow True σ^T/ϵ

$$\sigma_{ii}^e = P/A_0$$

$$\epsilon_{ii} = \Delta l/l_0$$

$$\sigma_{ii}^T = P/A = P/A_0 \cdot A_0/A = \sigma_{ii}^e \cdot l/l_0 = \sigma_{ii}^e \cdot (1 + \epsilon_{ii})$$

$$d\epsilon_{ii} = dl/l \rightarrow \epsilon_{ii} = \int_{l_0}^l d\epsilon_{ii}$$

$$= \int_{l_0}^l \frac{dl}{l}$$

$$= \ln \frac{l}{l_0} = \ln \frac{l_0 + \Delta l}{l_0}$$

$$= \ln(1 + \epsilon_{ii})$$

$$\epsilon_{ii} = \ln(1 + \epsilon_{ii})$$

ϵ_{ii} is too small,

$$\epsilon_{ii} \approx \epsilon_{ii}$$

(elastic strain \approx true strain)

(d) Max. uniform elongation before necking

= displacement until UTS

$$\sigma_{ii}^T = P/A \rightarrow P = \sigma^T \cdot A \leftarrow dP = 0 \text{ at necking}$$

$$dP = d\sigma^T \cdot A + \sigma^T \cdot dA = 0$$

$$-dA/A = d\sigma^T/\sigma^T$$

$$dV = 0 = d(A \cdot l)$$

$$= dA \cdot l + A \cdot dl = 0$$

$$dl/l = -dA/A$$

$$d\sigma^T/\sigma^T = dl/l = d\epsilon_{ii}$$

$$d\sigma^T/d\epsilon_{ii} = \sigma^T \leftarrow \text{constitutive law: } \sigma_{ii} = k(\epsilon_{ii})^n$$

$$\frac{d}{d\epsilon_{ii}} (\sigma^T) = \sigma^T$$

$$k \cdot n \cdot \epsilon_{ii}^{n-1} = k \cdot \epsilon_{ii}^n$$

$$\therefore n = \epsilon_{UTS}$$

at necking! why?

$$dP = 0$$

before necking : $\epsilon = \ln(1+e)$

$\rightarrow (1+e) = \exp(\epsilon)$

$\therefore e = \exp(\epsilon) - 1$ **Ans.**

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	strain hardening exponent	n	
A	0.26	0.2969	✓
B	0.05	0.0513	
C	N/A	N/A	

no necking (brittle)

(e) Max. work to necking

Let's assume ductility dominates material's behavior

$$U = U^T = \int_0^{\epsilon} \sigma^T d\epsilon \leftarrow \sigma^T = k\epsilon^n$$

$$= \int_0^{\epsilon} k\epsilon^n d\epsilon$$

$$= \frac{k}{n+1} \epsilon^{n+1} \Big|_0^{\epsilon} = \frac{k \cdot \epsilon^{n+1}}{n+1} = \frac{\sigma^T \cdot \epsilon}{n+1} \quad **$$

$\epsilon_{UTS} = n$

$\sigma^T = \sigma_{UTS} \cdot (1+e)$

$\epsilon_{UTS} = \ln(1+e_{UTS})$

$\exp\{\epsilon_{UTS}\} = 1+e_{UTS}$

$e_{UTS} = \exp\{\epsilon_{UTS}\} - 1 = \exp\{n\} - 1$

$\therefore \sigma^T = \sigma_{UTS} \cdot (1+e) = \sigma_{UTS} (1 + \exp\{n\} - 1) \rightarrow U_{UTS} = \frac{\sigma_{UTS} \cdot \exp\{n\} \cdot n}{n+1}$ **Ans.**

$\sigma_{11}^T = \frac{P}{A} = \frac{P}{A_0} \cdot \frac{A_0}{A} = \sigma_{11}^e \cdot \frac{l_0}{l} = \sigma_{11}^e \cdot (1+e_n)$

$d\epsilon_{11} = \frac{dl}{l} \rightarrow \epsilon_{11} = \int_{l_0}^l d\epsilon_{11}$

$= \int_{l_0}^l \frac{dl}{l}$

$= \ln \frac{l}{l_0} = \ln \frac{l_0 + \Delta l}{l_0}$

$= \ln(1+e_n)$

A: $\frac{\sigma_{UTS}}{n+1} = \frac{50 \cdot e^{0.26} \cdot 0.26}{0.26+1} = 13.38$

B: $\frac{320 \cdot e^{0.05} \cdot 0.05}{0.05+1} = 16.02$ ✓

C: no necking (brittle)

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f) Max. work (to fracture)

Let's assume ductility dominates material's behavior

$$U_f^T = \frac{\sigma_f^T \cdot \epsilon_f}{n+1}$$

$$\epsilon_f = \ln A_0/A$$

$$u = U^T = \int_0^{\epsilon} \sigma^T d\epsilon \leftarrow \sigma^T = k\epsilon^n$$

$$= \int_0^{\epsilon} k\epsilon^n d\epsilon$$

$$= \left[\frac{k}{n+1} \epsilon^{n+1} \right]_0^{\epsilon} = \frac{k \cdot \epsilon^{n+1}}{n+1} = \frac{\sigma^T \cdot \epsilon}{n+1}$$

$$RA\% = \frac{A_0 - A}{A_0} \times 100 = x$$

$$\epsilon_u = \ln \frac{A_0}{A} = \ln \frac{A_0}{A}$$

$$A_0 - A = A_0 \cdot x$$

$$A_0(1-x) = A$$

$$A_0/A = \frac{1}{1-x}$$

$$\therefore x = \frac{RA\%}{100}$$

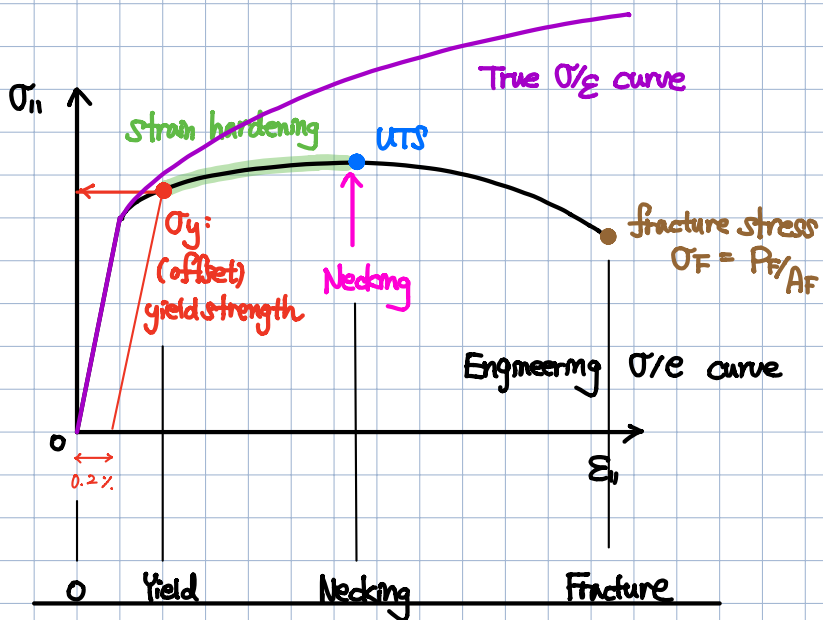
$$\epsilon_f = \ln A_0/A = \ln \left\{ \frac{1}{1-x} \right\}$$

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Max. work to fracture (= U_f^T)

$$A \quad \frac{(100) \cdot \ln \left\{ \frac{1}{1-0.65} \right\}}{0.26+1} = 83.32 \quad \checkmark$$

$$B \quad \frac{(350) \cdot \ln \left\{ \frac{1}{1-0.05} \right\}}{0.05+1} = 54.17$$



$\sigma = P/A_0$

$e = \Delta l/l_0$

$\sigma^T = P/A$

$\epsilon = \ln(1+e)$

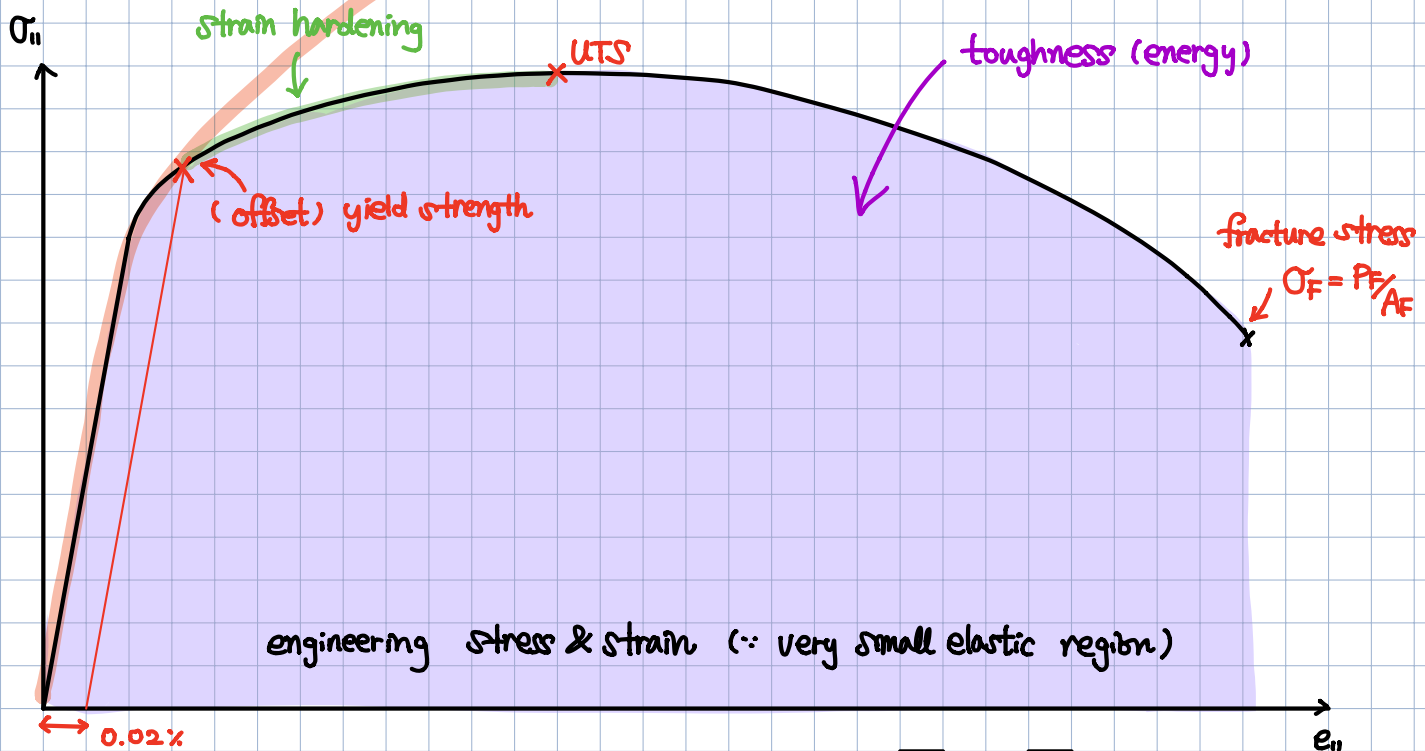
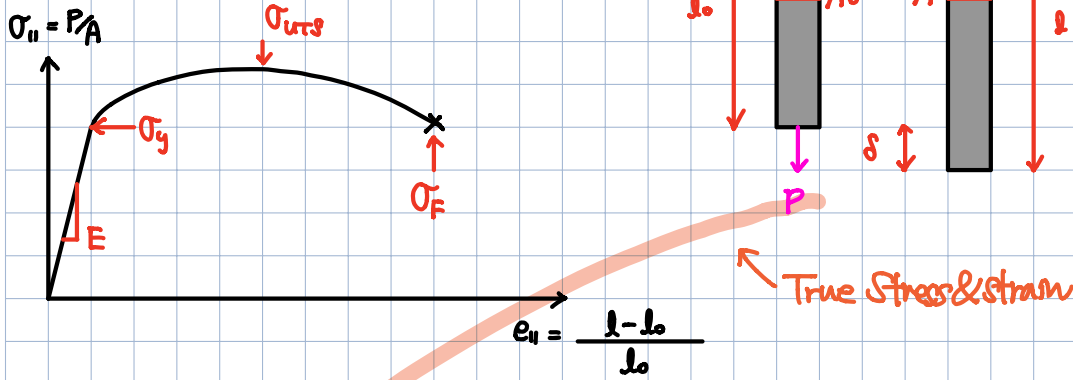
$\sigma^T = \sigma$ $\sigma^T = \sigma(1+e)$

$\epsilon^T = \epsilon$ $\epsilon^T = \ln A_0/A$

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Plasticity

uniaxial stress-strain curve



• **stiffness** : elastic modulus

• **strength** : resistance to plastic deformation
yield stress

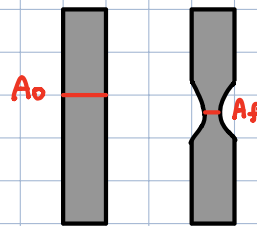
UTS (tensile strength) = P_{max}/A_0

• **ductility** : % elongation (total elongation = strain at failure)

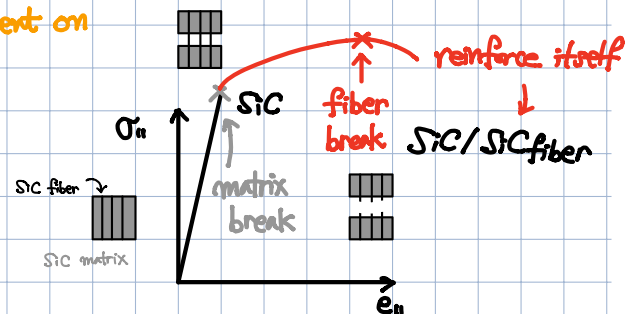
= $\frac{l_f - l_0}{l_0} \times 100\%$ ← can be dependent on gauge length

• % reduction in area = $\frac{A_0 - A_f}{A_0} \times 100\%$

• **toughness** : area of stress-strain curve



$\epsilon_u \sim 30-100\%$



(True) Stress: $\sigma_{ii}^T = P/A$ \longleftrightarrow (Engineering) stress: $\sigma_{ii}^e = P/A_0$

(True) Strain: $d\varepsilon_{ii} = dl/l$ (Engineering) strain: $\varepsilon_{ii} = \Delta l/l_0$

$\varepsilon_{ii} = \int_{l_0}^l d\varepsilon_{ii} = \int_{l_0}^l dl/l = \ln(l/l_0)$

$\varepsilon_{ij} = \varepsilon \left\{ \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \right\}$

$\varepsilon^{elastic} \ll \varepsilon^{plastic}$ plastic deformation
 \rightarrow assume elastic strains are negligible.

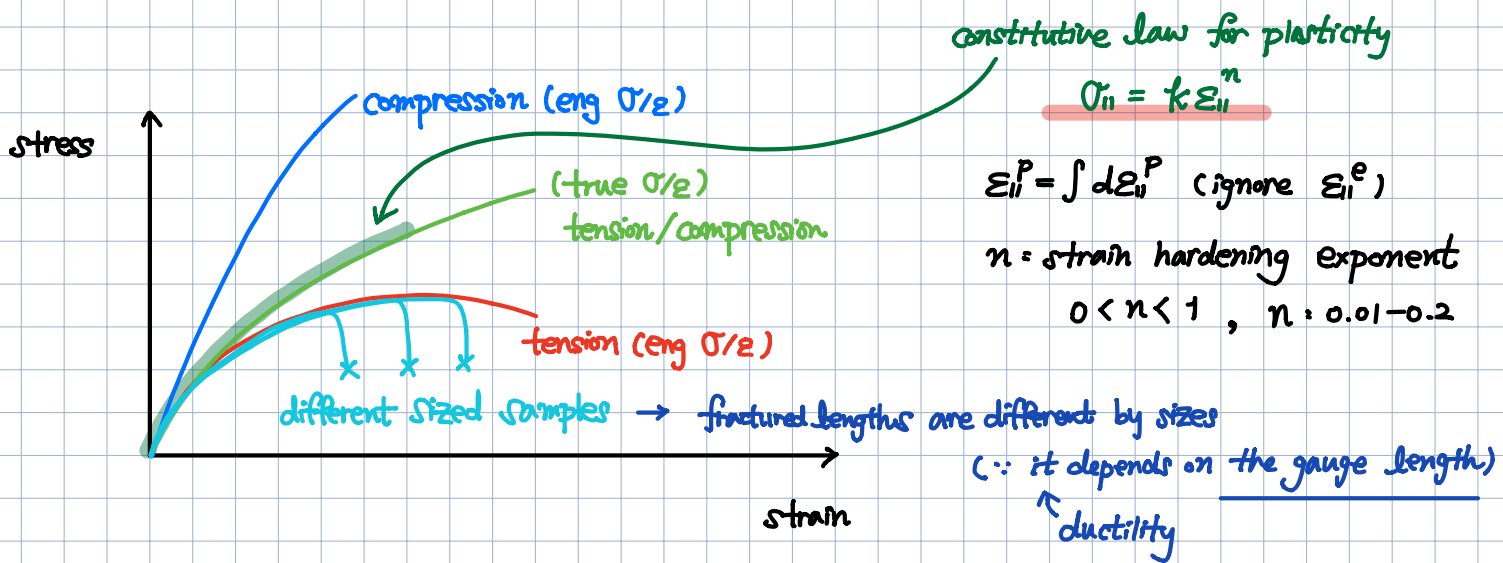
plastic deformation occurs without changing in volume. $\rightarrow \Delta V/V = 0, \nu = 1/2$

$V = A \cdot l = A_0 \cdot l_0 \rightarrow l/l_0 = A_0/A = 1 + e$ $\varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33} = \varepsilon = \Delta V/V$
 $V = A \cdot l, dV = 0 \rightarrow A dl + l dA = 0$
 $dl/l = -dA/A = d\varepsilon_{11}$

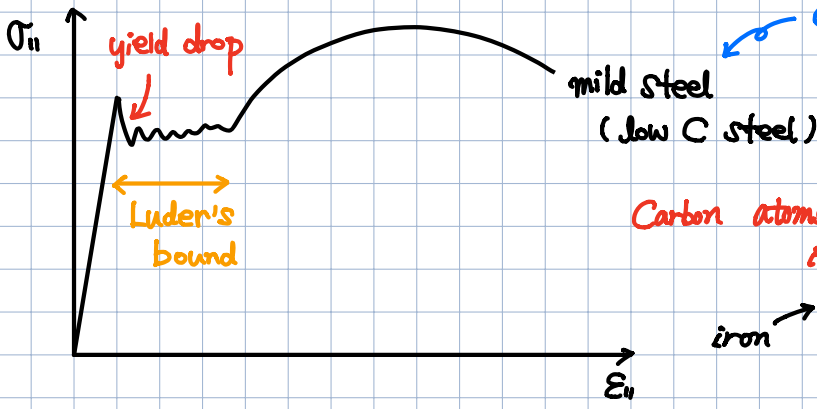
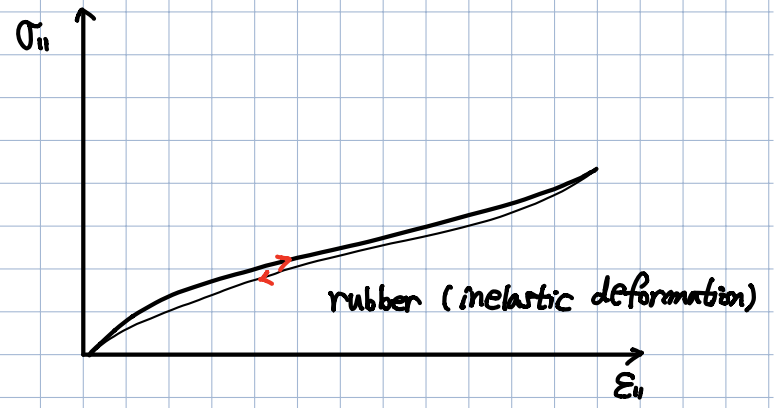
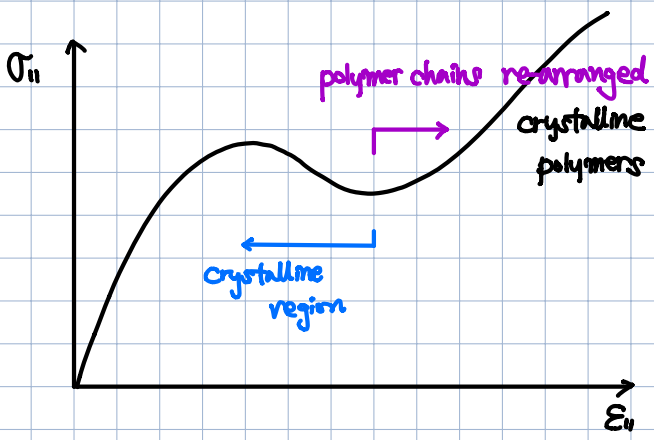
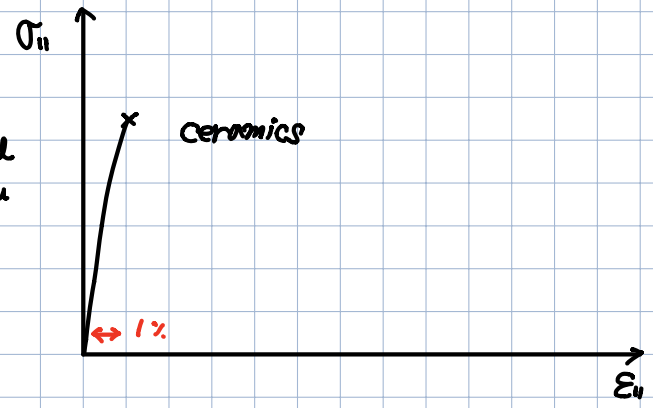
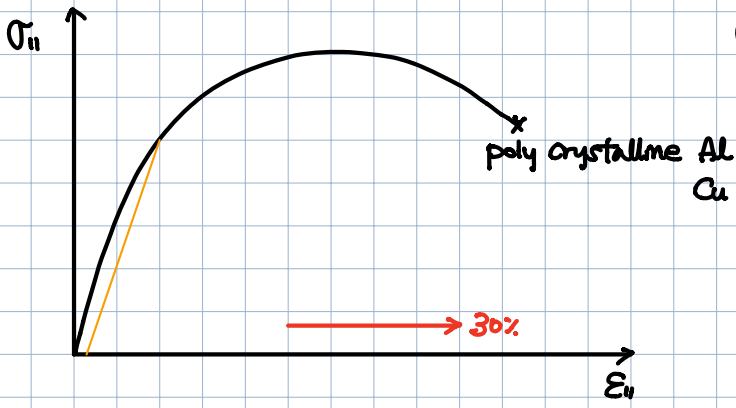
$\sigma_{ii}^T = P/A = P/A_0 \cdot A_0/A = \sigma_{ii}^e (1 + e_{ii})$

$\varepsilon_{ii} = \int_{l_0}^l dl/l = \ln(l/l_0) = \ln(1 + e_{ii})$ $\leftarrow x \approx \ln(1+x), \text{ when } x \text{ is small}$
 $e_{ii} \text{ is too small, } \varepsilon_{ii} \approx \varepsilon_{ii}$
 Elastic strain \approx plastic strain.

\rightarrow up to maximum load (necking)



$\varepsilon_{ii} = k' \sigma_{ii}^N, 1 < N < \infty$ (mechanical eng.)



Car bodies = you don't want yield drop

Carbon atoms = very small interstitial element

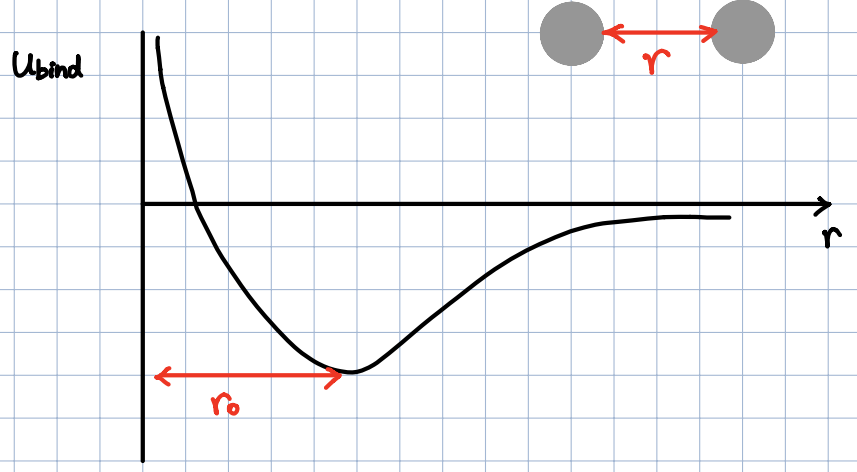
iron

carbon

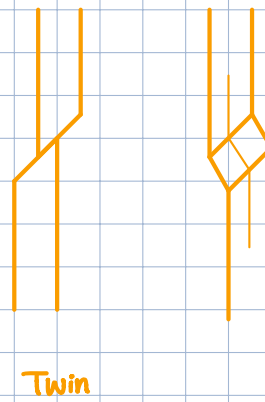
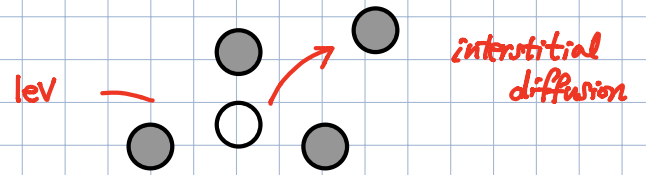
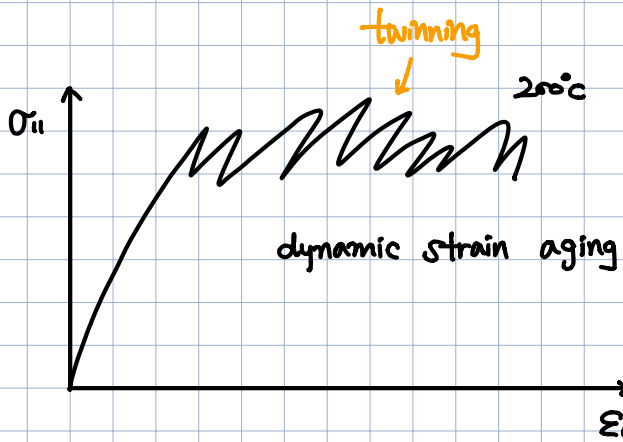
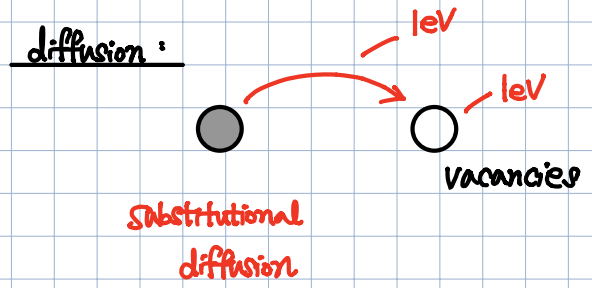
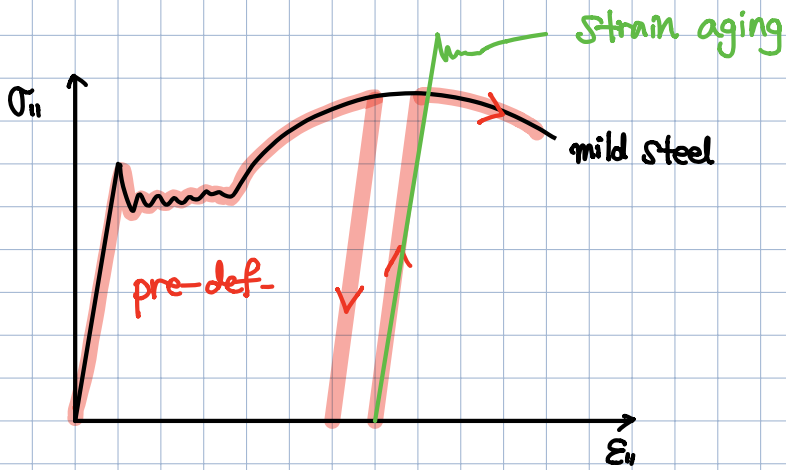
edge dislocation

carbon atmosphere

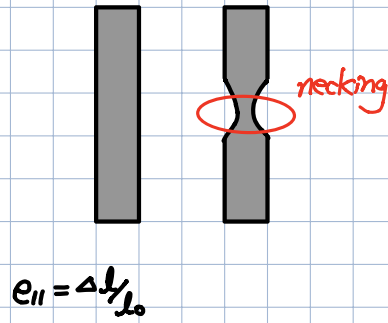
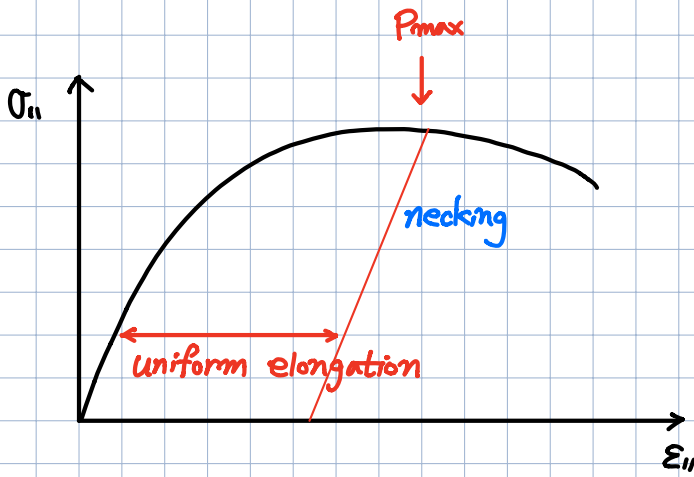
solid solution hardening



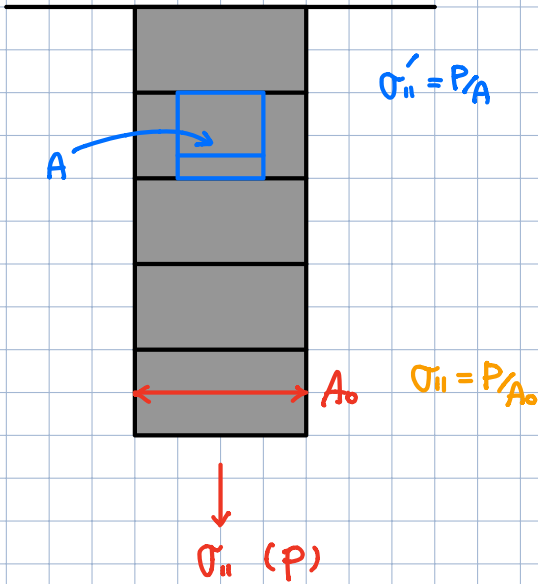
pre-deform the steel (rolling)



Necking (Plastic instability)



Why does necking occur?



(atomic preferences, ...)
for any reasons, A becomes smaller than A_0

as $A < A_0 \rightarrow \sigma'_{11} = \sigma_{11}$
 ↑ work hardening
 ↑ geometric softening

uniform elongation \rightarrow natural competition between strain hardening & geometric softening

\rightarrow Necking condition is $dP = 0$

uniaxial tension : $\sigma_{11} = P/A$

$P = A\sigma_{11}$

$dP = A d\sigma_{11} + dA \sigma_{11} = 0$

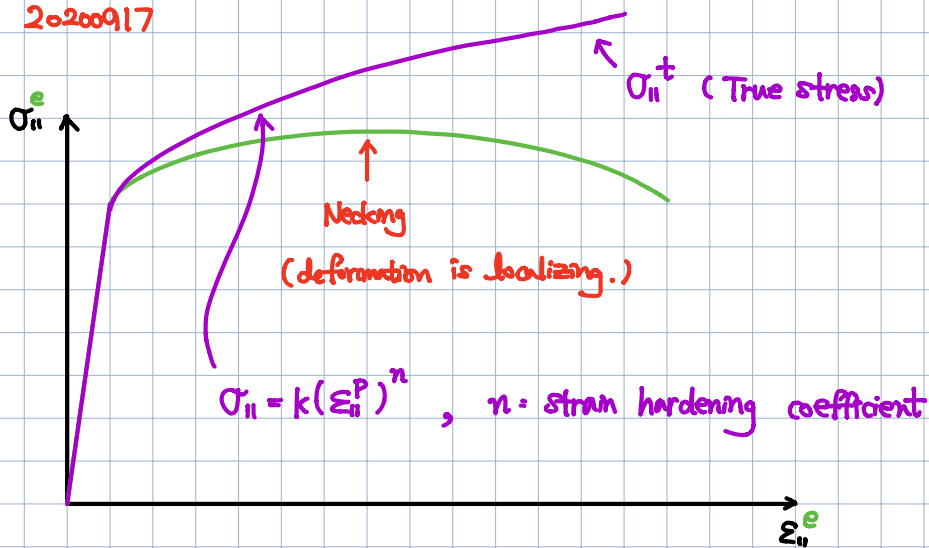
$\therefore \frac{d\sigma_{11}}{\sigma_{11}} = -\frac{dA}{A}$
 $= \frac{dl}{l} = d\epsilon_{11}$

$\leftarrow \epsilon_{11}^e \ll \epsilon_{11}^p, \quad \frac{dl}{l} = -\frac{dA}{A}$
 $\Delta V/V = 0$

$\frac{d\sigma_{11}}{d\epsilon_{11}} = \sigma_{11}$ at necking (considering relationship)

$\epsilon_{11} = n$ at necking

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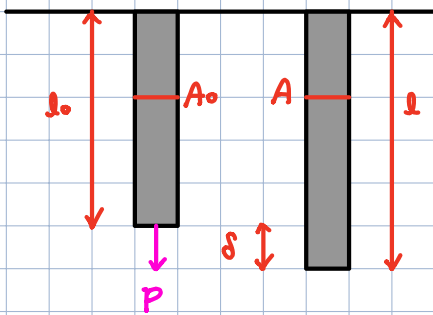


$$\sigma_{II}^e = P/A_0$$

not realistic
∴ it doesn't reflect
current situation

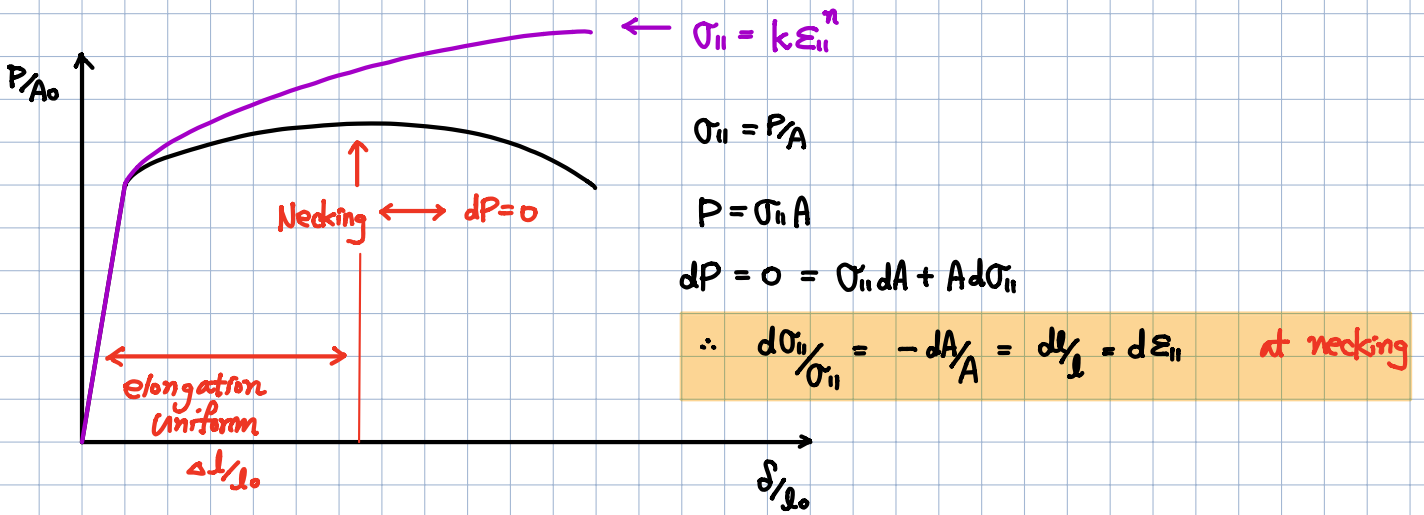
$$\sigma_{II} = P/A = \sigma_{II}^e (1 + e_{II})$$

$$\epsilon_{II} = \int d\epsilon_{II} = \ln l/l_0 = \ln(1 + e_{II})$$



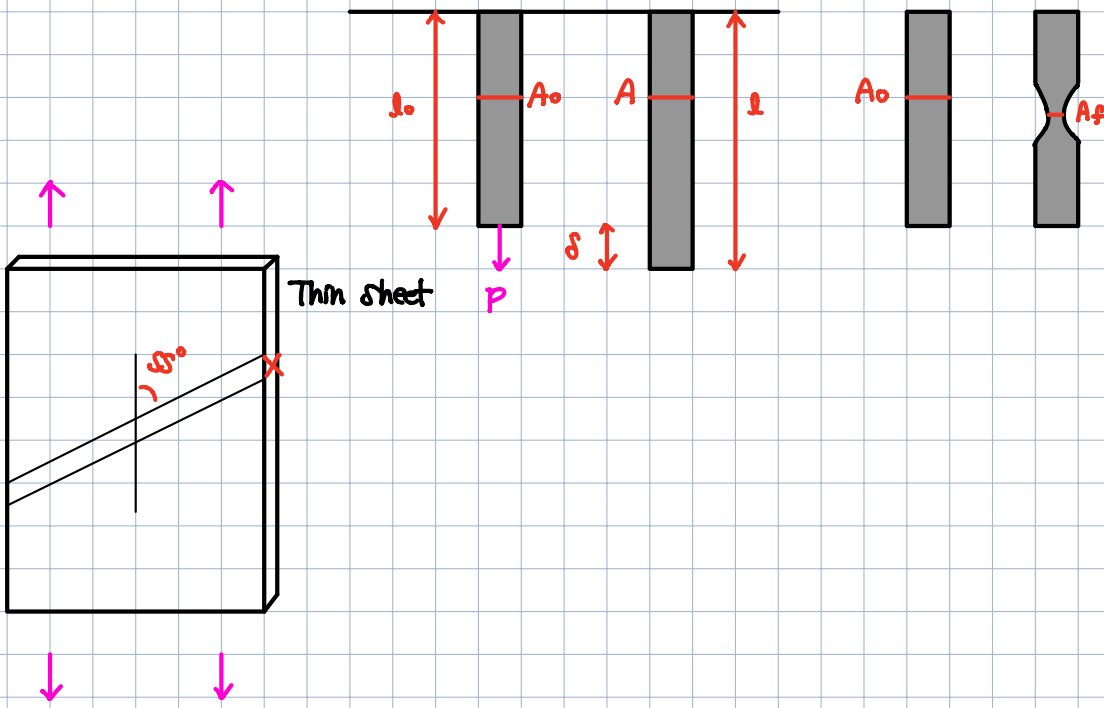
$$\epsilon_{II}^e \ll \epsilon_{II}^p \rightarrow \Delta V/V = 0 \rightarrow dl/l = -dA/A = d\epsilon_{II}$$

$$l/l_0 = A_0/A = 1 + e_{II}$$



(diffuse necking)

Necking criteria $\rightarrow d\sigma_{ii}/d\epsilon_{ii} = \sigma_{ii}$
 (consider criteria) Ans.,



necking criteria: $d\sigma_{ii}/d\epsilon_{ii} = \sigma_{ii}$

$\rightarrow d\sigma_{ii}/d\epsilon_{ii} = k \cdot n \cdot \epsilon_{ii}^{n-1} = \sigma_{ii} = k \cdot \epsilon_{ii}^n$

constitutive law: $\sigma_{ii} = k \epsilon_{ii}^n$

$\therefore \epsilon_{ii} = n$ at necking Ans.,

strain-induced $\gamma \rightarrow \alpha'$ (martensite)

- TRIP transformation-induced plasticity
- TWIP twinning-induced plasticity

$n \sim 0.2$

low C (mild steel)

$n \sim 0.02$

high C alloy steel \leftarrow high strength

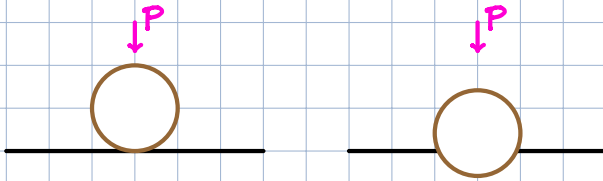
$n \sim 0.5$

austenitic stainless steel
 18Cr-8Ni

Hadfield Mn steel

13Mn steel

Hardness ↔ Strength



- Brinell hardness = $\frac{P}{\text{surface area of indent}}$ = stress (kg/mm^2) = BHN
 ↑
 Ball
 hardness (Brinell) $\sim 3 \times \text{UTS}$

- Meyer hardness = $\frac{P}{\text{projected area of indent}}$

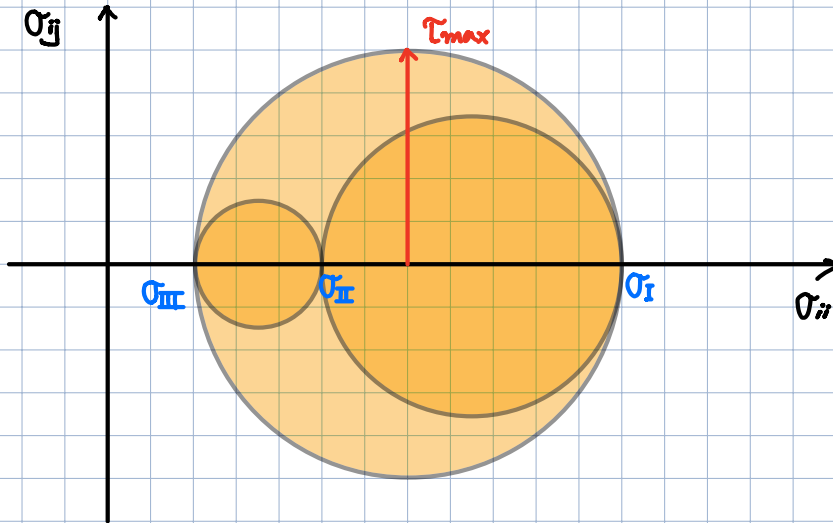
- Vicker's hardness: diamond, pyramid indenter
 (cubic corner — nanoindentation)

- Superficial hardness: Rockwell hardness — cone or ball

Material	ν	E (GPa)	σ_y (MPa)	σ_{UTS} (MPa)
Mildsteel	0.3	210	350	420
High strength alloy steel	0.3	210	2500	2200
Al	0.3	70	40	
Al-4%Cu	0.3	70	400	
diamond		1000		
Wood	0.04-0.5			
Ceramic Si_3N_4		320	16-30	
bone		15-20	100	

Yield Criteria for initial yielding

Tresca → initial yielding starts when max. shear stress > shear yield stress (k)



$$\tau_{max} = \frac{\sigma_I - \sigma_{III}}{2} = k$$

Mises → equivalent stress ($\bar{\sigma}$) = yield strength in tension y

$$\bar{\sigma} = \left[\frac{1}{2} \{ (\sigma_{11} - \sigma_{22})^2 + (\sigma_{22} - \sigma_{33})^2 + (\sigma_{33} - \sigma_{11})^2 \} + 3(\sigma_{12}^2 + \sigma_{23}^2 + \sigma_{31}^2) \right]^{1/2}$$

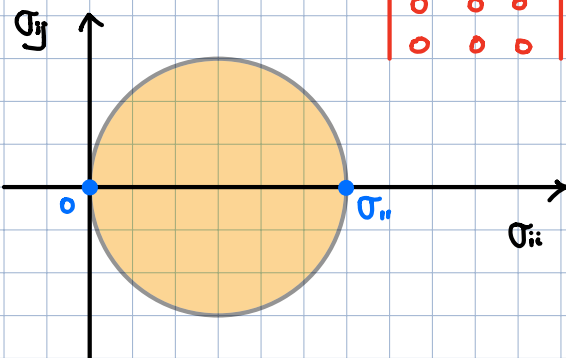
$$= \left[\frac{1}{2} \{ (\sigma_I - \sigma_{II})^2 + (\sigma_{II} - \sigma_{III})^2 + (\sigma_{III} - \sigma_I)^2 \} \right]^{1/2}$$

15' $\frac{1}{2}$ % max difference

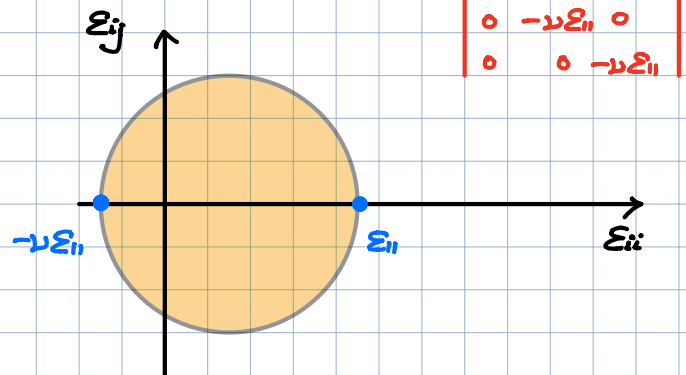
$$y = 2k \text{ (Tresca)}$$

$$= \sqrt{3} k \text{ (Mises)}$$

Uniaxial Tension



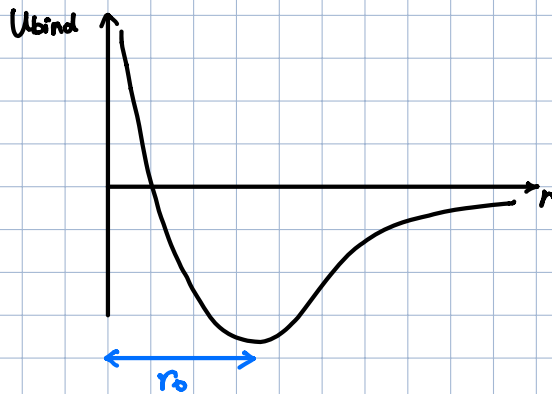
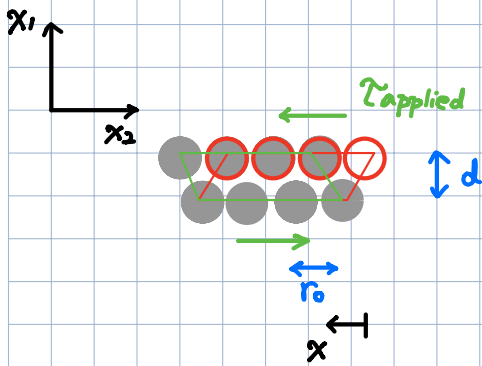
$$\begin{vmatrix} \sigma_{11} & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{vmatrix}$$



$$\begin{vmatrix} \epsilon_{11} & 0 & 0 \\ 0 & -\nu\epsilon_{11} & 0 \\ 0 & 0 & -\nu\epsilon_{11} \end{vmatrix}$$

plasticity $\nu = \frac{1}{2}$

Ideal or theoretical shear strength



Constant
↓

$$\tau \sim k \cdot \frac{\sin 2\pi x}{r_0}$$

for small x , $\frac{\sin 2\pi x}{r_0} \sim \frac{2\pi x}{r_0}$

$$\tau = \frac{k 2\pi x}{r_0}$$

$$\gamma = 2\epsilon_{12} \sim x/d \rightarrow \tau = G\gamma \sim Gx/d$$

$$\rightarrow Gx/d \sim \frac{k 2\pi x}{r_0} \rightarrow k \sim \frac{G r_0}{2\pi d}$$

$$\therefore \tau = \frac{G r_0}{2\pi d} \cdot \frac{\sin 2\pi x}{r_0} \quad \text{Ans.}$$

$$\hookrightarrow \tau_{\max} = G/2\pi \quad \text{when } \frac{\sin 2\pi x}{r_0} = 1$$

$d \sim r_0$ (order)

$$\gamma_{Th} = 2 \tau_{Th} \sim G/5 \quad : \text{Theoretical strengths}$$

2 orders of magnitude

→ defects in materials ↑
 ↓

$$\tau_{Th} \gg \tau_{crp} \quad \because \text{dislocations}$$

$$\tau_{Th}^{cohesive} > \tau_f$$